



Definition of parameters for a test section for the analysis of natural convection and coolant loss in the AP1000 nuclear reactor by similarity laws and fractional scaling analysis

L. F. S. Cadiz^a; M. A. B. da Silva^a; F. R. de A. Lima^a; C. A. B. de Oliveira^a

^a Departamento de Energia Nuclear - UFPE, 50.740-545, Cidade Universitária, Pernambuco, Brasil lfscadiz@gmail.com

ABSTRACT

The present work develops and analysis the main parameters of a test section for natural convection in case of failure of the pumping system as much as the loss of coolant in refrigeration accidents. For this realization a combination of laws of basic similarity and an innovative scale methodology, known as Fractional Scaling Analysis (FSA), was developed. The depressurizing is analyzed when a rupture occurs in one of the primary system piping of the AP1000 nuclear reactor. This reactor is developed by Westinghouse Electric Co., which is a PWR (Pressurized Water Reactor) with an electric power equal to 1000MW. Such a reactor is provided with a passive safety system that promotes considerable improvements in the safety, reliability, protection and reduction of costs of a nuclear power plant. The FSA is based on two concepts: fractional scale and hierarchy. It is used to provide experimental data that generate quantitative evaluation criteria as well as operational parameters in thermal and hydraulic processes of nuclear power plants. The results were analyzed with the use of computational codes.

Keywords: AP1000, LOCA, FSA.

1. INTRODUCTION

Loss of Coolant Accident (LOCA) or a cooling pump stop are serious accidents that can occur in a nuclear reactor and, if not controlled safely, the results can result in severe damage, especially in the core. A new methodology was developed by Zuber [1]. To promote scale analyzes in time-dependent processes involving an aggregate of modules and interactive processes (such as nuclear power plants), as well as integrating and organizing information and data of interest to complex structures and security analysis projects. This methodology is known as Fractional Scaling Analysis (FSA). It is used for the supply of experimental data that generate quantitative criteria and evaluation, as well as operational parameters in thermal and hydraulic processes of nuclear power plants. In engineering projects, carrying out tests with structures presenting true magnitude, that is, prototypes, involves quite high costs, making it difficult or even impossible to carry out such tests. Thus, a test section is designed for the natural convection process and the loss of coolant the AP1000 reactor, using scaling analyzes.

2. MATERIALS AND METHODS

The techniques used to analyze the safety of nuclear reactors, specifically with respect to thermal hydraulic behavior, have as mains objective to analyze the real operational conditions to quantitatively predict the operational conditions. In the nuclear reactor in full operation, there are numerous factors that can cause it to enter a state of emergency. The temperature, pressure and flow rate of the coolant are fundamental properties to be controlled. They can range from poor performance to a major accident. The AP1000, developed by Westinghouse Electric Co, is a PWR (Pressurized Water reactor) type nuclear reactor with an electric power of 1000MW. Such a reactor is endowed with a passive safety system that promotes considerable improvements in the safety, reliability, protection and cost reduction of a nuclear power plant, simplicity being its key concept. Similarity consists of obtaining common dimensionless groups, a projected structure and the model used. Dimensional analysis is used as a tool.

The FSA compares model and prototype, or between two or more test facilities in the study of the same phenomenon. In determine a region of space characterized by a variable V, which undergoes a change caused by an agent Φ , then, according to the FSA:

$$\Phi = \frac{\mathrm{d}V}{\mathrm{d}t} \tag{1}$$

The fractional rate of change, ω , of variable V is defined:

$$\omega = \frac{1}{V} \frac{dV}{dt} = \frac{\Phi}{V}$$
(2)

V where is the variable contained in a space and submitted to the variation agent. Considering a reference value V_0 , the variation suffered by the variable, δV , is quantified and thus the fractional or metric variation of V effect is defined by:

$$\Omega = \frac{\delta V}{V_0} \tag{3}$$

I can also be obtained by means of Eq.(3), as:

$$\Omega = \omega \cdot \delta t = \frac{\Phi}{V_0} \delta t \tag{4}$$

Analyzing the equations, processes are similar where have their state variable altered by the same fractional quantity, that is, they have the same values of the effect metric. In this new methodology, similarity requires only the equality of the values of Ω , without the equalities of the other parameters ω and δt [1]. Thus, the fractional scaling can be applied at any level of interest or complexity.

For application of the fractional scaling analysis the Eq.(5), represents the rate of depressurizing of a fluid in volume V of a system [2].

$$\frac{\mathrm{d}p}{\mathrm{d}t} = \frac{1}{\mathrm{VK}_{\mathrm{s,sys}}} \left[-\sum \dot{V}_{j} + \frac{\mathrm{v}_{\mathrm{fg}}}{\mathrm{h}_{\mathrm{fg}}} \left(\dot{Q}_{2\phi} \right)_{\mathrm{liq}} + \sum_{j=\mathrm{l,v}} \left(\frac{\beta_{j}}{\rho_{j}.\mathrm{C}_{\mathrm{pj}}} \dot{Q}_{\mathrm{liq}} \right) + \left(\frac{\beta}{\rho_{.}\mathrm{C}_{\mathrm{p}}} \right) \mathrm{P}_{\mathrm{B}} + \frac{\gamma-1}{\gamma} \frac{\dot{Q}_{\mathrm{N}2}}{p} \right] = \sum_{j} \Phi_{j} \tag{5}$$

In this equation the individual rates are defined for contraction or volumetric expansion, \dot{V}_j , and the individual pressure variation agents, Φ_j . The term $K_{s,sys}$ represents the isentropic compressibility of the system, which is given by the sum of the individual isentropic compressibilities weighted by the respective volumetric fractions. Each term of variation of Eq.(5) is normalized, so that its dimensionless variable ae of unity order. If Y_0 is the initial reference values for the variation agent Y(t), the normalized term has the form:

$$Y^{+}(t) = \frac{Y(t)}{Y_{0}}$$
(6)

The volumetric variation rates, $\dot{V}_1(t)$, are now dimensioned according to Eq.(7)

$$\dot{V}_{j}^{+} = \frac{\dot{V}_{j}(t)}{\dot{V}_{j}(t)}$$
 (7)

By combining with Eqs.(5), (6) and (7), Eq.(8) is obtained, the terms of which are defined in Table 1:

$$\frac{dp^{+}}{dt} = \omega_{r}\Phi_{r}^{+} + \omega_{2\phi}\Phi_{2\phi}^{+} + \omega_{l}\Phi_{l}^{+} + \omega_{v}\Phi_{v}^{+} + \omega_{P_{B}}\Phi_{P_{B}}^{+} + \omega_{N_{2}}\Phi_{N_{2}}^{+}$$
(8)

In addition to having great utility in the synthesis of data for the system, the fractional rates o variation, ω , provide the scale criterion for projects and classify the variation agents, Φ , according to their importance. The coefficients are shown in Eq.(8), multiplied by reference time, t_{ref} , to give the fractional effect metrics, $\Omega_j = \omega_j \cdot t_{ref}$, which must have the same values in all installations where the same phenomenon happens. It is observed that there will only be similarity in the behavior of the pressures over time, if each normalized agent also has the same values in the model and in the prototype [2]. Table 1 shows the normalized fractional variation agents and the initial rates of fractional variation.

Variation agents	Agents in the range of	Initial Fractional Rates of		
	pressure variation: Φ_j^+	Change: w _j		
Flow through rupture (j=1)	$\Phi_r^+ = -\frac{\dot{V}_1^+}{K_{s,sys}^+}$	$\omega_{\rm r} = \frac{\left(\dot{\rm V}_1\right)_0}{K_{\rm s,sys}}$		
Phase change (j=2)	$\Phi_{2\phi}^{+} = \frac{\left(v_{fg}/h_{fg}\right)^{+}}{K_{s,sys}^{x}}\dot{Q}_{2\phi}^{x}$	$\omega_{2\phi} = \frac{\left(v_{fg}/h_{fg}\right)_{0}}{V\Delta p\left(K_{s,sys}\right)_{0}}\dot{Q}_{2\phi}$		
Expansion or single-phase thermal contraction (j=3,4)	$\Phi_{l,v}^{+} = \frac{\left(\beta/(\rho c_{p})\right)_{l,v}^{+}}{K_{s,sys}^{+}}$	$\omega_{l,v} = \frac{\left(\beta/(\rho c_p)\right)_{l,v}}{V\Delta p\left(K_{s,sys}\right)_0} \dot{Q}_{l,v}$		
Expansion due to pump heating (j=5)	$\Phi_{P_{B}}^{+} = \frac{\left(\beta/(\rho c_{p})\right)_{l}^{+}}{K_{s,sys}^{x}} P_{B}^{+}$	$\omega_{P_{B}} = \frac{\left(\beta/(\rho c_{p})\right)_{l,0}}{V\Delta p\left(K_{s,sys}\right)_{0}}(P_{B})$		
Expansion of inert gas due to heating on the wall (j=6)	$\Phi_{N_2}^+ = \frac{\dot{Q}_{N_2}}{p^+ + (p_{amb}/\Delta p)}$	$\omega_{Q_{N_2}} = \frac{\gamma - 1}{\gamma} \frac{\dot{Q}_{N_2}}{V(\Delta p)^2 (K_{s,sys})_0}$		

Table 1: Standardized fractional variation agents and initial rates of fraction variation.

Where, $\dot{Q}_{l,v}$, $\dot{Q}_{2\phi}$, \dot{Q}_{N_2} , and P_B represent, respectively, heat transfers for one phase, for the two phase, for inert gas (Nitrogen) and pumping power. The superscript terms "+" represent the dimensionless terms. The terms K_{sys} , v_{lv} , h_{lv} , β , ρ , C_p and γ represent respectively isentropic compressibility, the difference between specific volumes of vapor and liquid, vaporization enthalpy, isobaric thermal expansion coefficient, density of water, the specific heat at constant pressure and the isentropic exponent. The dimensionless time has the following form:

$$t^+ = |\omega_r|t$$

(9)

Eq.(8) can be converted to:

$$\frac{dp^{+}}{dt^{+}} = \Phi_{r}^{+} + \sum_{j=2}^{6} \widehat{\omega}_{j} \Phi_{j}^{+}$$
(10)

The ratios of the fractional variation rates corresponding to the pi groups of the traditional scale methods [2] are defined at the beginning of the LOCA by:

$$\widehat{\omega}_{j} = \frac{\omega_{j}}{|\omega_{r}|}, j=2...6, \left|\widehat{\omega}_{j}\right| < 1, \, \widehat{\omega}_{1} = \widehat{\omega}_{r} = 1 \tag{11}$$

Even so, this procedure does not scale the time correctly for the whole household. For a correct time scale, all the variation agents must be added. Thus, Eq.(10) is divided by the fractional rate of change of the aggregate given by the sum of the individual rates, according to Eq.(12):

$$\overline{\omega} = \omega_{\rm r} + \sum_{j=2}^{6} \omega_j \tag{12}$$

The effective dimensionless time of the whole aggregate is given by the effect metric, defined by:

$$\mathbf{t}^{+} = |\Omega| = |\overline{\omega}|\mathbf{t} \tag{13}$$

Since the fractional rates of change present in Eq.(8) represent initial rates under conditions of maximum power, and considering that the system is in the steady state, when the occurrence of the rupture, all the variation agents are balanced, constituting the rupture the only exception, making, in the special case of a loss of coolant, the fractional rate of change of the aggregate coincides with that of the rupture. To normalize Eq.(8), the normalization of each volumetric rate of change should be used, dividing it by that of the aggregate, as explained by Eqs.(14) and (15).

$$\dot{V}_{j}^{+} = \frac{\dot{V}_{j}(t)}{\left|\Sigma_{j} \dot{V}_{j}\right|_{t=0}}$$
(14)

Replacing Eq.(14) in Eq.(8) leads to Eq.(15), in which the term Φ_j^+ represents each normalized variant:

$$\frac{\mathrm{d}p^{+}}{\mathrm{d}t} = \frac{\left|\Sigma_{j}\dot{V}_{j}\right|_{t=0}\Sigma_{j}\dot{V}_{j}}{V\Delta p(K_{s,sys})_{0}K_{s,sys}^{+}} = \frac{\left|\Sigma_{j}\omega_{j}\right|}{K_{s,sys}^{+}} = \left|\overline{\omega}\right|\sum_{j}\varphi_{j}^{+}$$
(15)

However, as defined in Eq.(14), the division of Eq.(15) by the absolute value of the fractional rate of change of the aggregate, term before the summation on the right side of this equation, results in the correct dimensionless time for the process. Thus, the non-dimensional pressure variation rate is defied by Eq.(16):

$$\frac{dp^{+}}{dt^{+}} = \sum_{j=1}^{6} \phi_{j}^{+}$$
(16)

Following the dimensional analysis, which originated the similarity scale criteria, there are significant the existence of significant dimensionless groups or numbers representing momentum, mass and energy transfer characteristics in the single-phase flow under a natural circulation regime. These criteria are assumed to be of relative importance at the time of the establishment of the small-scale experiment. Thus, it can reproduce more similar effects of prototype, in study, of real scale.

The experimente has as representative numbers of the phenomenon that is desired to simulate the following dimensionless numbers:

• Richardson number:

$$R = \frac{g\beta\Delta T_0 l_0}{u_0^2} \tag{17}$$

• Stanton number:

$$St = \left(\frac{4hl_0}{\rho C_p u_0 d}\right)$$
(18)

3. RESULTS AND DISCUSSION

The main parameters of a test section are determined in order to simulate the loss of coolant in the AP1000 reactor core. The numerical simulation of the depressurizing for a pressure vessel containing only water, with the geometric and thermodynamic parameters of such a reactor was performed. For a first study, the area of rupture of 0.005m² is adopted



Figure 1: Comparison between the model and the simulated prototype

The agreement of the curves in Figure 1 shows the effectiveness of the FSA method. The main parameters obtained by this work, which faithfully represent the temporal variation of the pressure in a depressurizing process for a future construction of a test section, show the similarity bet ween the model and the prototype by the FSA. The effect metric (Ω) for both is 0,400. Applying the aforementioned procedures to other conditions, by varying areas of ruptures, obtain the results presented in Table 2.

	Breaking	Volume	$\dot{V}_{rc}(0)$	$\Delta PxK_{sys}(0)$	ω ₁	S _t	Ω
	area (m ²)	(m ³)	(m³/s)		(s ⁻¹)		
Prototype	10x10 ⁻³	30,9556	0,1672	0,0119	2,0069	1	0,0238
Model 1	8x10 ⁻³	0,1433	0,1338	0,0109	1,6055	0,3862	0,0238
Model 2	6x10 ⁻³	0,1433	0,1003	0,0109	1,2042	0,5149	0,0238
Model 3	4x10 ⁻³	0,1433	0,0669	0,0109	0,8028	0,7723	0,0238
Model 4	2x10 ⁻³	0,1433	0,0334	0,0109	0,4014	1,5446	0,0238

Table 2: Scale parameters for other models subject to the same initial pressure and temperature conditions, but with variation of the area of rupture.

Figure 2: Comparison of the prototype and the different models, according to table 2.



Figure 2 shows the efficiency of the method. The curves overlap during the depressurizing process for models with different areas of ruptures, so by the FSA the similarity does not depend on the area of rupture, considering the limitations of scale. The same procedures were performed; however, the volume of the model was varied. Table 3 details such results. The graph of pressure and time dimensioned by the FSA is given in Figure 3.

	Breaking	Volume	$\dot{V}_{rc}(0)$	$\Delta \mathbf{PxK}_{\mathbf{sys}}(0)$	ω ₁	S _t	Ω
	area (m ²)	(m ³)	(m³/s)		(s ⁻¹)		
Proto- type	10x10 ⁻³	30,9556	0,1672	0,0119	2,0069	1	0,0238
Model 1	5x10 ⁻³	0,1033	2,3x10 ⁻⁴	0,0109	0,4650	0,3170	0,0238
Model 2	5x10 ⁻³	0,1533	2,3x10 ⁻⁴	0,0109	0,3133	0,4704	0,0238
Model 3	5x10 ⁻³	0,2033	2,3x10 ⁻⁴	0,0109	0,2363	0,6239	0,0238
Model 4	5x10 ⁻³	0,2533	2,3x10 ⁻⁴	0,0109	0,1896	0,7773	0,0238

Table 3: Scaling parameters for other models subject to the same initial pressure and temperature conditions, but with volume variation.





Analyzing the results, it is noticed that even with the variation of the volumes, there is similarity between prototype and model, during the process of depressurizing. This was evidenced by Figure 3. Different volumes of the model and areas of ruptures are simulated and by the synthesis presented by the FSA similarity is obtained. It is proved that by the FSA, the similarity of this process does not depend on the volume of the adopted model, considering the limitations of scale. The effect matric

values are similar, which are similar for the prototype and the model, indicating, therefore, similar depressurizing, regardless of the experiment times and the dimensions being different. Only the contribution of the volumetric flow due to rupture is considered, the non-dimensional fractional variation rate is unitary for both the model and the prototype:

$$\widehat{\omega}_1 = \frac{\omega_1}{\overline{\omega}} = \frac{\omega_1}{\omega_1} = 1 \tag{19}$$

Similarity for natural convection the determination of the best combinations between the design parameters and the operational conditions will constitute a series of objectives to be maximized or minimized. Table 4 shows the result of the dimensionless numbers selected to represent the phenomenon of natural convection, through similarity.

	Richardson	Stanton
Prototype	0,6182	4,3152x10 ⁻⁴
Model	0,6139	4,3135x10 ⁻⁴

Table 3: Comparison of dimensionless numbers between prototype and model.

4. CONCLUSION

The investigations carried out in the present study showed the feasibility of proposed methodology for small scale thermohydraulic experiments for simulating a natural circulating regime or a loss of coolant. This methodology supports the decisions to be taken by the designers during the design phase, and allows to obtain, in this way, more efficient and economical experiments.

5. ACKNOWLEDGMENT

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